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## Research Article

# **Effect of Installation of Solar Collector on Performance of Balcony Split Type Solar Water Heaters**

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The influences of surface orientation and slope of solar collectors on solar radiation collection of balcony split type solar water heaters for six cities in China were analyzed by employing software TRNSYS. The surface azimuth had greater effect on solar radiation collection in high latitude regions. For deviation of the surface slope angle within  $\pm 20^{\circ}$  around the optimized angle, the variation of the total annual collecting solar radiation was less than 5%. However, with deviation of 70 $^{\circ}$  to 90 $^{\circ}$ , the variation was up to 20%. The effects of water cycle mode, reverse slope placement of solar collector, and water tank installation height on system efficiency were experimentally studied. The thermal efficiencies of solar water heater with single row horizontal arrangement all-glass evacuated tubular collector were higher than those with vertical arrangement at the fixed surface slope angle of 90 $^{\circ}$ . Compared with solar water heaters with flat-plate collector under natural circulation, the system thermal efficiency was raised up to 63% under forced circulation. For collector at reverse slope placement, the temperature-based water stratification in water tank deteriorated, and thus the thermal efficiency became low. For improving the system efficiency, an appropriate installation height of the water tank was suggested.

#### 1. Introduction

With the challenges of fossil fuel depletion and global warming, utilization of renewable energy attracted extensive attention over the whole world. In the past twenty years, solar water heaters were widely used due to their low cost and simplicity in technology. The key component of solar water heater, the solar collector, gathers solar radiation and converts it into heat energy. The solar collector is usually installed on building's roof with a surface slope, which is equal to the local geographical latitude with an addition of 10 degrees. However, with development of urbanization, more and more high-rise building rose in the city. There is not enough room on buildings' roofs for all inhabitants to install their own solar water heaters. Moreover, when the hot water flows through the overlong pipes from heat water tanks on roofs to the users' rooms, the hot water would cool and even freeze in winter. So balcony split type solar water heater is proposed and plays a very important role in applications combined with high-rise buildings in recent years.

The collectors of balcony split type solar water heaters are usually attached on high-rise building external walls or balcony rails. The collectors do not always face south, the optimal orientation, but vary according to the building orientations. With consideration of light shadow to rooms and building beauty, it is difficult for collectors to be installed in optimal surface slope. The amount of heat collected by solar collectors relates closely with the collector surface slope. Many researches on the optimum orientation and tilt angle of solar collectors in some countries or regions, such as Brunei Darussalam, Syria, and Saudi Arabia, have been performed [1-4]. Gunerhan and Hepbasli [5] built a model to calculate the optimum tilt angles for which the total radiation on the collector surface was a maximum for a specific period and recommended that the solar collector was mounted at the monthly average tilt angle and the slope was adjusted once a month. Haitao and Yanfeng [6] analyzed the heat energy collected instantly by the evacuated tube solar collector with different slope angles in LHASA and considered the best angle in this area is 46°. Ong et al. [7] studied the performance of U-tube solar water heaters mounted on walls and balconies and concluded that their performance depended largely upon their orientation. For U-type evacuated glass tube solar collector fixed vertically on the balcony wall, the mean daily collector efficiency is about 40%, and the solar fraction is satisfied in summer and autumn [8]. Besides, the influences of size and configuration of the water tank [9, 10], collectors with colour absorber [11-13], and fluid flow and heat transfer [14-16] on the system performance were also studied. Hobbi and Siddiqui [17] modelled an indirect forced circulation solar water heating system with a flat-plate collector and optimized the design parameters by TRNSYS simulation program. The designed system could provide 83-97% and 30-62% of the hot water demands in summer and winter in Montreal of Canada, respectively. Seveda [18] performed some experiments on a natural circulation closed thermosyphon flat-plate solar water heater in sunny and cloudy days in India and improved the performance of the system by using glycol as working fluid. With the black ceramic coatings with solar absorptance of 0.93-0.97, the thermal efficiency of an allceramic solar collector is, respectively, 47.1% and 50% when the solar thermal collectors act as balcony railings and on building roofs [19]. Souliotis et al. [20] developed integrated collector storage (ICS) type solar water heaters. Chien et al. [21] developed a two-phase thermosyphon solar water heater with the best efficiency of 82% in experiments, higher than the conventional solar water heaters. Ong and Tiwari [22-24] developed simulation methods and test performance of solar water heating systems under the thermosyphon mode between the collectors and the storage tank.

In this paper, the Transient System Simulation Program, TRNSYS [25, 26], was utilized to analyze the effects of surface orientation and surface slope of solar collectors on solar radiation collection in six typical cities of China with different geographic latitudes. TRNSYS, developed by the University of Wisconsin, is an effective tool for predicting the performance of solar water heaters. Afterwards, two sets of balcony split type solar water heaters with flat-plate collectors and two sets of balcony split type solar water heaters with all-glass evacuated tubular collectors were set up for experiments. The effects of the water cycle mode of the solar water heater, the reverse slope placement of the solar collector, and the installation height of the water tank on the efficiency of the solar heaters were studied.

# 2. Effect of Surface Orientation and Slope on Solar Radiation Collection

The TRNSYS Type 45 model is adopted to discuss the effects of azimuth angles and slope angles of solar collectors on solar radiation collection for the thermosyphon solar water heaters (TSWH). The required hourly meteorological data for all six cities were taken from the Typical Meteorological Year data bank. The simulation time step is set at 10 minutes.

2.1. Effect of Surface Azimuth on Solar Radiation Collection. Figure 1 shows the effects of surface azimuth of solar collectors on solar radiation collection in six cities of China with different geographic latitudes (geographic locations of

TABLE 1: Geographic location of six cities in China.

City name	Geographic location			
City manie	East longitude	North latitude		
Qiqihar	123.55	47.22		
Shanghai	121.48	31.22		
Guangzhou	113.23	23.16		
Beijing	116.46	39.92		
Kunming	102.70	25.07		
Haikou	110.35	20.02		

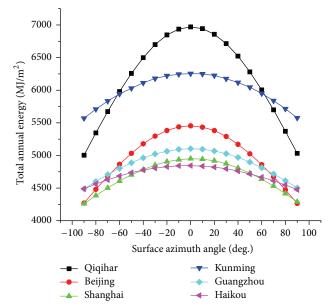


FIGURE 1: Effect of surface azimuth on collecting radiation in cities with different geographic locations.

the six cities are illustrated in Table 1 and Figure 2). Qiqihar and Beijing are located in higher northern latitude and have more significant change of the total annual solar energy with change of the azimuth, comparing with the other four cities. In Qiqihar, the total annual solar energy is 7000 MJ/m<sup>2</sup> and 5000 MJ/m<sup>2</sup>, respectively, when the solar collector faces the south and has the azimuth of 90°. The variety is up to 2000 MJ/m<sup>2</sup>. In Beijing, the total annual solar energy is 5300 MJ/m<sup>2</sup> and 4200 MJ/m<sup>2</sup>, respectively, when the solar collector faces the south and has the azimuth of 90°. The variety is 1100 MJ/m<sup>2</sup>. Haikou is with the lowest northern latitude, and the curve of surface azimuth's effect on solar radiation collection for Haikou is the most flat. The total annual solar energy is 4800 MJ/m<sup>2</sup> and 4500 MJ/m<sup>2</sup>, respectively, when the solar collector faces the south and has the azimuth of 90°. The variety is only 300 MJ/m<sup>2</sup>. This means that the solar radiation collection is affected by the surface azimuth of the solar collector. The higher the latitude of the place is, the greater the effect of the surface azimuth of the solar collector on solar radiation collection is.

Figure 3 shows the effects of surface azimuth of solar collectors with 25° slope angle on the total annual solar

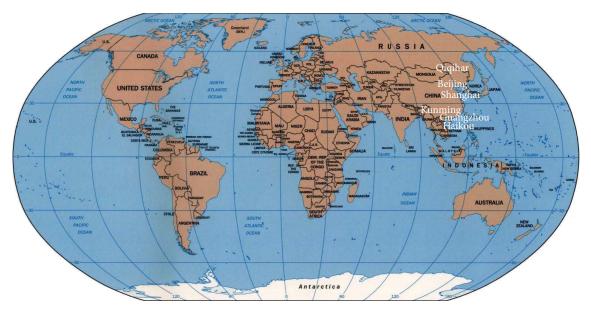
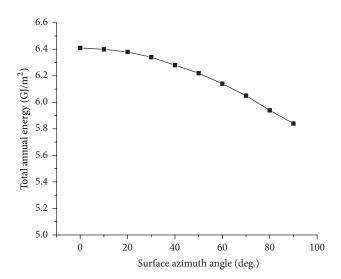


FIGURE 2: Location of six cities in China.



 $\ensuremath{\mathsf{Figure}}$  3: Effect of surface azimuth on solar radiation collection in Kunming.

radiation collection in Kunming, China. In Figure 3, the most total annual solar radiation collection is obtained with surface azimuth of  $0^\circ$ . The total annual solar radiation collection decreases with increase of the surface azimuth angle. When the surface azimuth angle changes from  $0^\circ$  to  $20^\circ$ , the variation of the total annual collecting solar radiation is less than 1%. However, with further increasing of the surface azimuth, the variance on the total annual collecting solar radiation becomes larger. When the surface azimuth increases from  $40^\circ$  to  $60^\circ$ , the change of the collecting solar radiation is around 2.3%. And the change is 3.3% when the surface azimuth varies from  $70^\circ$  to  $90^\circ$ . The total annual collecting solar radiation at the surface azimuth of  $90^\circ$  is 91.0% of that at the surface

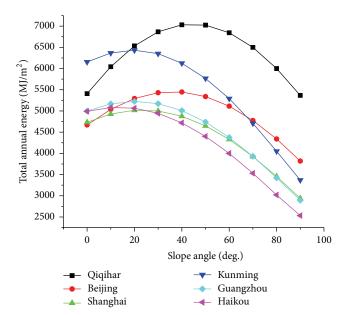


FIGURE 4: Effect of surface slope on annual solar radiation collection in cities with different geographic locations.

azimuth of  $0^{\circ}$ . Therefore, the surface azimuth has little effect on the collecting solar radiation in Kunming, especially when the surface azimuth is within  $20^{\circ}$  from the south orientation.

2.2. Effect of Surface Slope on Solar Radiation Collection. Figure 4 shows the effects of the surface slope of the solar collectors on annual solar radiation collection in six cities of China with different geographic locations. In the figure, the optimized surface slope angle is about the local geographic latitude minus 10° when the solar collector faces south. With

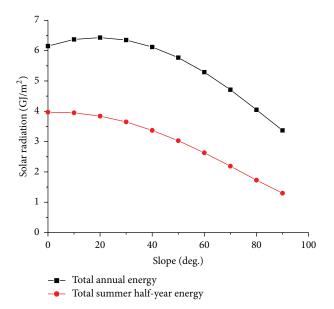


FIGURE 5: Effect of surface slope on solar radiation collection in Kunming.

a change of the surface slope angle within ±20° around the optimized surface slope angle, the variation of the total annual collecting solar radiation is less than 5%. However, with further increasing or decreasing of the surface slope angle around the optimized surface slope angle, the variety of the total annual collecting solar radiation becomes larger. With the deviation of the surface slope angle from the optimized surface slope angle from 20° to 40°, the change of the total annual collecting solar radiation is around 5%. With the deviation of the surface slope angle from the optimized surface slope angle from 70° to 90°, the change of the total annual collecting solar radiation is up to more than 20%. Therefore, the vertical installation of the solar collector is not proper from the view of solar radiation collection. With the comprehensive consideration of solar radiation collection, external wall load-bearing, light shadow to the lower rooms, and building beauty, the surface slope angle of 60° is appro-

In Figure 5, the optimal surface slope angle of the solar collector is 18° in Kunming when the collector surface faces south, and in this case, the most total annual collecting solar radiation of 6.43 GJ/m² is achieved. The variation of the collecting solar radiation is less than 5% with a change of the surface slope angle from the optimal angle within 20°. However, with a further increase/decrease of the surface slope angle, the influence of the surface slope angle on the total annual collecting solar radiation becomes significant. When the surface slope angle increases from 70° to 90°, the change of the collecting solar radiation is more than 20%. The total annual collecting solar radiation is 3.37 GJ/m² when the surface slope angle is 90° and is only 52.4% of the most total annual collecting solar radiation at the optimal surface slope angle.

The total collecting solar radiation of 3.97 GJ/m<sup>2</sup> in summer half year (from March 21 to September 23) is achieved in

Kunming when the surface slope angle of the solar collector is  $0^{\circ}$ . The total collecting solar radiation in summer half year is around  $1.3 \, \text{GJ/m}^2$  with the surface slope angle of  $90^{\circ}$  and is 32.7% of that at the surface slope angle of  $0^{\circ}$ . The total collecting solar radiation in summer half year is around  $1.4 \, \text{GJ/m}^2$  with the surface slope angle of  $70^{\circ}$  and is 55.1% of that at the surface slope angle of  $0^{\circ}$ . It was shown that, for balcony split type solar water heaters, the vertical installation of the solar collector would result in relatively little solar radiation collection. With comprehensive consideration of the solar radiation collection, the wall load-bearing, and the avoidance of light shadow to the rooms, the installation of the solar collectors with a slope angle of  $70^{\circ}$  is optimal. The above simulation results were in good agreement with some conclusions drawn by [27-29].

### 3. Experimental Setup

Two sets of balcony split type solar water heaters with flatplate collectors and two sets of balcony split type solar water heaters with all-glass evacuated tubular collectors were set up for experiments. The tilt angle of the collector bracket is adjustable and varies from 0 to 90. The total area of the flatplate collector in each set of balcony split type solar water heaters is 1.5 m<sup>2</sup>, and the effective radiation collection area is 1.325 m<sup>2</sup>. The frames of the flat-plate collectors are made from stainless steel plate, the thermal insulating material is rock wool with the thickness of 4 cm, and the cover plate is made from toughened glass with optical transmittance of 90%. The aperture area (the effective radiation collection area) of the all-glass evacuated tubular collector in each set of balcony split type solar water heaters is 1.317 m<sup>2</sup>. 10 evacuated collector tubes are placed side by side in each collector. The length of a single tube is 1800 mm, the outer glass tube diameter is 58 mm, and the inner glass tube diameter is 47 mm. The aluminum pipe, wrapped with the rubber insulation cotton, is employed to build the circulating pipeline. The cylindrical water tank, with a total volume of 120 L, is horizontally placed. Other parameters of the system are shown in Table 2.

The thermal performance testing system of solar water heaters TRM-2, made by Jinzhou Sunlight Technology Development Company Ltd., was employed in our experiments. The technical specifications of the test equipment are shown in Table 3. The experiments were performed in Kunming, China, with the geographic coordinate (25.02°N, 102.68°E) from January to May. The system automatic recording time step was set in 10 minutes in the experiments.

#### 4. Results and Discussions

4.1. Performance of Solar Water Heater with All-Glass Evacuated Tubular Collector at a Fixed Surface Slope Angle of 90° under the Natural Circulation. The thermal performance of the solar collector is one of the most important factors to decide the performance of solar water heater. The Hottel-Whillier equation is usually used for evaluating the thermal performance of the solar energy heat-collection plates:

$$Q_u = A_c F_R \left( G_T \left( \tau \alpha \right) - U_L \left( T_i - T_a \right) \right), \tag{1}$$

TABLE 2: Parameters of the solar water heaters.

	Parameters				
Items	Solar water heater with flat-plate collector	Solar water heater with all-glas evacuated tubular collector			
Aperture area of the collector (m <sup>2</sup> )	1.325	1.317			
$F_R( aulpha)$	0.76				
$F_R U_L \text{ (kJ/hr·m}^2 \cdot \text{K)}$	15				
Assemble angle of collector (degrees)	0~90	0~90			
Distance between water inlet and water outlet of collector (m)	0.86				
Distance between water outlet of collector and tank bottom (m)	0.14~1.34				
Length of cycle pipeline (m)	2~4.5	2~4.5			
Heat loss coefficient of cycle pipeline (kJ/hr·m²·K)	0.1	0.1			
Tank volume (m <sup>3</sup> )	0.12	0.12			
Total heat loss coefficient (kJ/hr·K)	5.0				

TABLE 3: Technical index of TRM-2 thermal performance testing system of solar water heaters.

	Measurement range	Measurement accuracy	Resolution	Other parameters
Temperature	−40°C~350°C	±0.1°C	0.1°C	
Ambient temperature	−40°C~70°C	±0.1°C	0.1°C	
Wind speed	0~60 m/s	$\pm (0.3 + 0.03) \mathrm{m/s}$	$0.1\mathrm{m/s}$	Threshold wind velocity $\geq 0.5 \text{ m/s}$
Solar radiation	$0\sim 2000  \text{W/m}^2$	$\pm 5\% \text{ W/m}^2$	$1  \text{W/m}^2$	

where  $Q_u$  is the useful gain of the solar energy heat-collection plate,  $W/m^2$ ;  $A_c$  is the heat-collection area of the solar energy heat-collection plate;  $F_R$  is the thermal conversion factor of the heat-collection plate;  $G_T$  is the solar radiation on the tilted surface;  $\tau$  is the transmissivity of the glass covering;  $\alpha$  is the absorptivity of the collector plate;  $U_L$  is the thermal transmittance from collector surface to ambient air;  $T_i$  is the collector surface temperature;  $T_a$  is the ambient air temperature.

The heat-collection efficiency of the heat-collection plate is defined as the ratio of the useful gain to the solar radiation on the solar energy heat-collection plate:

$$\eta = \frac{Q_u}{A_c G_T},\tag{2}$$

where  $\eta$  is the heat-collection efficiency of the heat-collection plate, %.

According to the arrangement mode of the evacuated tubes in the solar collector, the evacuated tubular collector can be classified as the single row vertical arrangement collector, the single row horizontal arrangement collector, and the double row horizontal arrangement collector. Some experiments were performed in winter (January) in Kunming by utilizing the balcony wall-mounted solar water heater with the single row horizontal arrangement collector and the single row vertical arrangement collector at a fixed surface slope angle of 90° to investigate their performances under the natural circulation. The experimental results are shown in Table 4.

In Table 4, the system thermal efficiencies of the solar water heater with the single row horizontal arrangement

all-glass evacuated tubular collector were higher than those with the single row vertical arrangement all-glass evacuated tubular collector at the fixed surface slope angle of 90°. With the initial water temperature around 13°C and the solar radiance around 16.5 MJ, the final water temperatures of the former reached around 40°C after 6 hours, the water temperature rise was 24.6-28.2°C, and the system thermal efficiencies were around 60%. However, with similar initial water temperature and similar solar radiance, the final water temperatures of the latter were only around 35°C after 6 hours, the water temperature rise was 19.8-22.1°C, and the system thermal efficiencies were below 49%. The main reason is, for the solar water heater with the single row vertical arrangement all-glass evacuated tubular collector, the cold fluid flows down to the bottom of the evacuated tube along the shaded side of the evacuated tube. In the process of flowing down, the cold fluid absorbs energy unceasingly and merges into the upwelling current before it reaches the bottom of the tube. That is to say, there is the short circuit of the fluid circulation, which results in the heat transportation deterioration of the system and reduces the system efficiency. In this work, the length of the all-glass evacuated tube is 1800 mm. To reduce the tubular length would be helpful for enhancing the heat transportation of the system and improving the efficiency.

4.2. Performance of Solar Water Heater with Flat-Plate Collector at a Fixed Surface Slope Angle of 90° under Natural Circulation and Forced Circulation. The size of the flat-plate collector is usually  $2 \, \text{m} \times 1 \, \text{m}$  or  $1.5 \, \text{m} \times 1 \, \text{m}$ , and its height is  $2 \, \text{m}$  or  $1.5 \, \text{m}$ . However, the height of balcony sidewall is

D.t.	Single row horizontal arrangement				Single row vertical arrangement			
Date	Jan. 24	Jan. 25	Jan. 28	Jan. 29	Jan. 30	Jan. 23	Jan. 24	Jan. 25
Ambient temperature (°C)	21.7	16.6	14.6	14.7	15.6	16.3	17.2	16.6
Average wind speed (m/s)	0.5	0.7	1.3	1.6	0.7	1.4	0.6	0.7
Irradiance (MJ)	17.7	15.8	17.2	16.4	17.1	17.0	17.4	15.8
Time (hr)	6.0	6.0	6.0	6.0	6.0	6.0	6.0	6.0
Initial temperature of tank (°C)	14.4	12.7	12.7	13.1	13.6	15.2	13.8	14.2
Final temperature of tank (°C)	40.7	37.3	39.7	39.6	41.8	36.4	35.9	34.0
Temperature rise (°C)	26.3	24.6	27.0	26.5	28.2	21.2	22.1	19.8
Tank volume (L)	120	120	120	120	120	120	120	120
Efficiency	0.565	0.595	0.600	0.617	0.629	0.475	0.483	0.480

TABLE 4: Performance of all-glass evacuated tubular collector at a surface slope angle of 90°.

Table 5: Performance of solar water heater with flat-plate collector at a fixed surface slope angle of 90° under natural circulation and forced circulation.

Date	Natural c	Forced circulation		
	Jan. 24	Jan. 25	Feb. 3	Feb. 4
Ambient average temperature (°C)	17.3	16.6	17.4	16.6
Average wind speed (m/s)	0.6	0.7	1.1	1.4
Irradiance (MJ)	17.6	15.8	15.7	15.3
Time (hr)	6.0	6.0	6.0	6.0
Initial temperature of tank (°C)	14.3	13.8	16.2	19.6
Final temperature of tank (°C)	24.4	22.9	42.3	44.8
Temperature rise (°C)	10.1	9.1	26.1	25.2
Efficiency	0.218	0.219	0.629	0.654
Useful energy (MJ)	5.08	4.56	13.12	12.64
Consumption of energy (MJ)	0.0	0.0	1.73	1.73

generally around 1 m. It is not feasible to mount directly the flat-plate collector on the balcony sidewall according to the conventional installation method. An alternative solution is to put the collector on its side, so its height would meet the requirement of the balcony sidewall.

But placing the collector on its side is not conducive to the natural circulation of the system and would ultimately affect the thermal efficiency of the system. Therefore, the forced circulation is necessary in some cases. Table 5 presents the experimental results of the solar water heater with the flatplate collector at a fixed surface slope angle of 90° under natural circulation and forced circulation. The pump power for the forced circulation is 80 W, and the mass flow rate is 40 kg/h.

In Table 5, the efficiency of the solar water heater with the flat-plate collector placed on its side at a fixed surface slope angle of 90° under natural circulation is only 21.8%. Compared with the efficiency of the solar water heater with the single row all-glass evacuated tubular collector at a fixed surface slope angle of 90° under the natural circulation in Table 4, the efficiency of the solar water heater with the flat-plate collectors is disappointing under the same conditions. It mainly results from the small diameter of the up-flow tube in the flat-plate collector; consequently, the flow

resistance is large, and the system circulation is not smooth. The utilization of the forced circulation may improve the circulation of the system. In similar irradiance and wind speed, the system thermal efficiency could be raised up to 63%, and the circulating pump power is only 13% of the heat gain.

4.3. Effect of Reverse Slope Placement of Evacuated Tubular Collector on Performance of Solar Water Heater under Natural *Circulation.* The driving force of the water circulation of the solar water heater under natural circulation originates from the thermosyphon pressure and is relatively smaller than that under water forced circulation. In designed installation, the water outlet of the collector is generally slightly higher than the water inlet, at least at the same horizontal level, to reduce resistance and increase thermosyphon pressure. That is to say, the collecting tube is placed in the form of climbing toward the water outlet. The structural characteristics of the flat-plate solar water heater usually ensure the down slope setup of the solar collector. For the all-glass evacuated tubular collector of split type solar water heater, the water outlet of the collector could be lower than the water inlet due to the uneven floor or an installation error in an actual installation process. In the circumstances, the water circulation direction

TABLE 6: Experimental performances of all-glass vacuum tube solar water heaters with collectors at down slope placement and at rever	se
slope placement.	

Date	Re	verse slope placement	Down slope placement		
	Feb. 3	Feb. 4	March 1	Jan. 29	Jan. 30
Ambient average temperature (°C)	17.1	16.8	20.3	14.8	15.6
Average wind speed (m/s)	1.2	1.4	0.3	1.6	0.7
Irradiance (MJ)	20.2	20.6	19.2	19.7	20.6
Time (hr)	6.0	6.0	6.0	6.0	6.0
Initial temperature of tank (°C)	27.7	14.3	16.9	12.0	19.6
Final temperature of tank (°C)	49.9	38.7	40.7	45.2	43.6
Tank volume (L)	120	120	120	120	120
Temperature rise (°C)	22.2	24.5	23.8	33.2	24.0
Efficiency	0.419	0.453	0.471	0.641	0.617

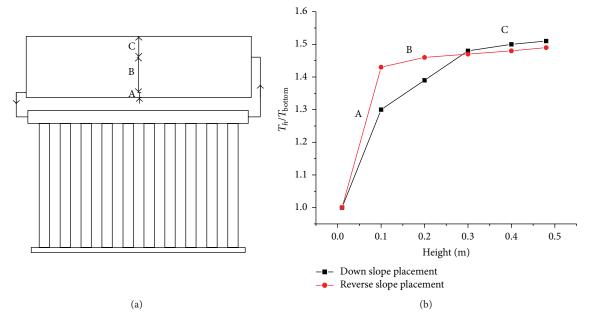


FIGURE 6: Temperature-based water stratification in water tank for collectors at down slope placement and at reverse slope placement.

of the natural circulation solar water heater would change, which consequently affects the mixing way of the cold water and the hot water and the water stratification in water tank. The performance of the solar water heater is degraded.

Table 6 shows the experimental performances of all-glass vacuum tubular solar water heaters with collectors at down slope placement and at 5° angle reverse slope placement. In similar solar irradiance, ambient temperature, and wind speed, the collecting efficiency of solar water heaters with collector at down slope placement is above 60%; however, the collecting efficiency of solar water heaters with collector at reverse slope placement is about 45%, 15% lower than that of the former.

The performance deterioration of the solar water heater with collector at reverse slope placement results from the change of the temperature-based water stratification in water tank. In Figure 6(a), the water tank is divided into three

regions from the bottom to the top of the tank: region A, region B, and region C. Region A is from the tank bottom to the water outlet of the tank. With the region, the deposition on the tank bottom could be avoided to enter the collector. The water in region A does not circulate into the collector, and it is actually a stagnant water area. Region B is from the water outlet to the water inlet of the tank. Region C is from the water inlet to the tank top.

In Figure 6(b),  $T_h/T_{\rm bottom}$  rises linearly in region A for collector at both the down slope placement and the reverse slope placement. The water temperature distribution in the water stagnant area is due to the heat transfer method: heat conduction only. The heat input in the region originates from the water in region B. In region B, the water temperature has a linear rise with the increase of the height when the collector is at down slope placement. In this case, the water circulation direction is as shown in Figure 6(a), and the hot water from

the collector enters the water tank from the inlet on the upper part. The higher temperature water is on the upper part and the lower temperature water is on the lower part, which would not result in the flow of the water in the region. So the temperature-based water stratification forms. The water outlet for users is on the upper part of the tank; therefore, with the down slope placement, the hot water is preferentially supplied to users. However, for collector at reverse slope placement, the water temperature varies slightly in region B, and the water stratification is worse than the former. In this case, the water circulation direction is reverse with that shown in Figure 6(a) and the hot water from the collector enters the water tank from the inlet on the lower part. The higher temperature water is on the lower part and the lower temperature water is on the upper part, which would result in the water mix in the region. The temperature-based water stratification is not clear. In region C, the buoyancy results in the rise of hot water and the water mix, so the water temperature varies gently for collector at both the down slope placement and the reverse slope placement. From the above analysis, for collector at the reverse slope placement, the temperature-based water stratification in water tank is poor, and the collection efficiency is low. The solar collector should avoid reverse slope placement.

4.4. Effect of Installation Height of the Water Tank on Efficiency of the Solar Heater. For balcony split type solar water heater, the height difference between the water tank and the solar collector is the driving force of the natural water circulation. Too small height difference would result in shortage of the circulation driving force. On the contrary, too large height difference would increase the length of the pipeline and the cooling area, thus lowering the thermal efficiency of the water heater. Therefore, the modest height difference is essential to achieve enough circulation pressure and simultaneously keep the high thermal efficiency of the water heaters as much as possible and improve water temperature stratification in the water tank. Figure 7 shows the effects of the height of the water tank on the efficiency of the solar heater.

In Figure 7, when the height difference between the bottom of the water tank and the outlet of the collector varied from 0.14 m to 0.74 m, the thermal efficiency of the solar water heater increased from 62.5% to 67.7%. The rise mainly resulted from the increment of the thermal siphon pressure greater than the increment of the resistance of ducting in the stage. The boost of the water circulation would timely bring the heat collected by the solar collector to the water tank, thereby increasing the system thermal efficiency. However, with a further increase of the height difference from 0.74 m to 1.34 m, the thermal efficiency reduced from 67.7% to 64%. In this stage, while the rise of the installation height of the water tank would magnify the thermal siphon pressure, it also increases greatly the resistance of ducting and the cooling area of the system, eventually leading to the fall of the system thermal efficiency instead of the rise. That is to say, there is an optimal height difference between the bottom of the water tank and the outlet of the collector for the thermal efficiency of the system to reach its maximum. For the solar water heater with a total solar collecting area of 1.5 m<sup>2</sup> and

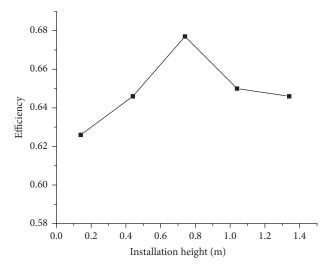


FIGURE 7: Effect of the height of the water tank on the efficiency of the solar heater.

a water tank capacity of  $120\,L$  (as discussed in the work), the optimal height difference is around  $0.74\,m$ . In the actual installation, the height difference of  $0.44\,m\sim1.04\,m$  according to particular case is considered as reasonable, because the change of the system thermal efficiency in this range does not exceed 3%.

#### 5. Conclusions

TRNSYS was employed to analyze the effect of surface orientation and slope of solar collectors on solar radiation collection of balcony split type solar water heaters. In regions with high latitude, the surface azimuth of solar collector had greater effect on solar radiation collection. For deviation of surface slope angle within  $\pm 20^\circ$  around the optimized angle, the variation of the total annual collecting solar radiation was less than 5%. However, with deviation of 70 $^\circ$  to 90 $^\circ$ , the variation was up to 20%.

In our experiments, the system thermal efficiency of solar water heater with single row horizontal arrangement all-glass evacuated tubular collector was higher than those with single row vertical arrangement at the fixed surface slope angle of 90°. Under forced circulation, the system thermal efficiency of solar flat-plate water heater was raised up to 63% compared with that under natural circulation, and the circulating pump power was only 13% of the heat gain. For collector at reverse slope placement, the temperature-based water stratification in water tank deteriorated, and thus the thermal efficiency became low. The height difference between water tank and solar collector had influence on the thermal siphon pressure and the resistance. For improving the system efficiency, an appropriate installation height of the water tank was suggested.

#### **Conflict of Interests**

The authors declare that there is no conflict of interests regarding the publication of this paper.

#### Acknowledgment

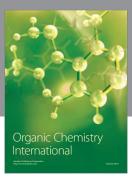
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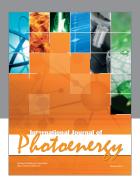
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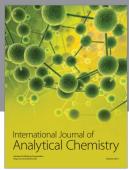
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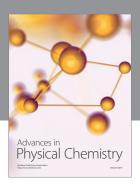
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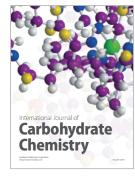
















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